

Excellence In Fluid Control

# **Hydraulic Cylinders**

- Standard Cylinders
- Tie Rod Cylinders
- Custom Built Cylinders





# **Company Profile**

Microtact a leading name in Hydraulic Valves and Systems worldwide, has deliveries across the globe. Microtact has good knowledge of the needs of the customers, with a tradition of quality and service that spans many decades.

We are a ISO 9001 Company, ensuring the quality standards as per International Standards and Specifications.

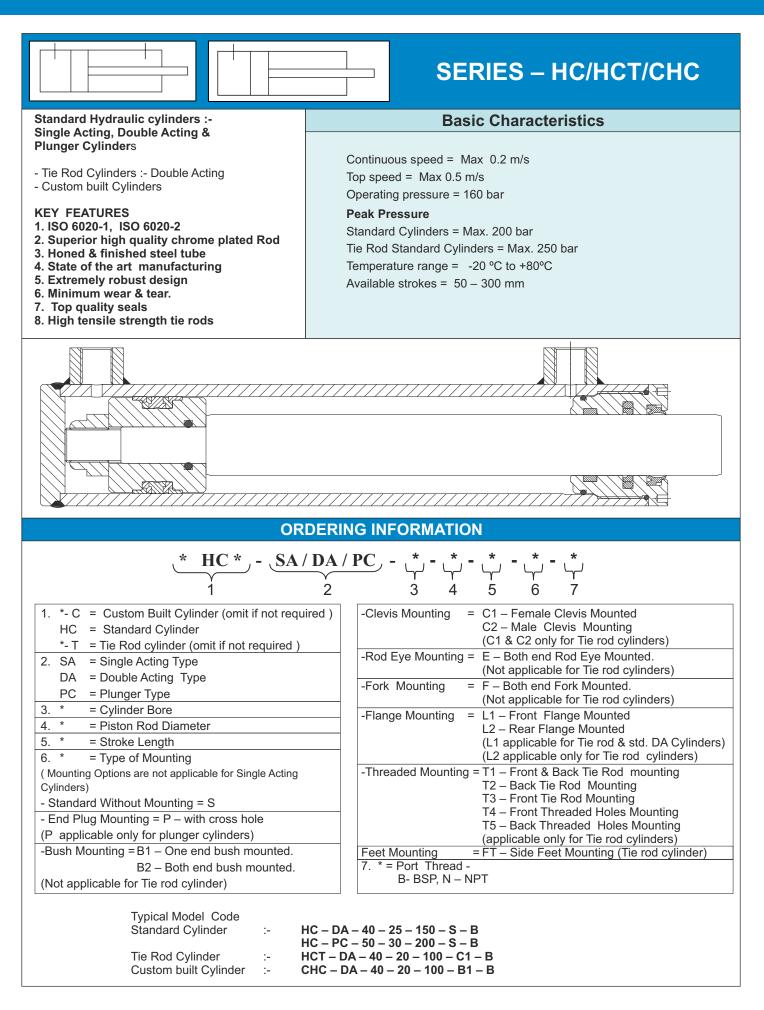
We at Microtact, guarantee our customers the international experience, reliability and back-up in providing solutions which are both most effective and cost-effective.

Microtact products means more than Three Decades of experience, innovative product development, high quality standards, application know-how and prompt service back-up to all customers.

# Infrastructure







Microtact Hydraulic cylinders are designed and manufactured according to the standards: **ISO 6020-1**, **ISO 6020-2** All cylinders manufactured according to these standards have a unified installation measurement as per the stroke to ease replacement. Cylinders are primarily intended for the automization of work processes (processing machines and other servohydraulic drives), and they are also the best solution for systems where a simple exchangeability of components is required since the cylinders are designed modularly. When designing them special attention is paid to simple servicing without special tools that makes servicing faster and more economical.

A hydraulic cylinder is the actuator or "motor" side of a system. The "generator" side of the system is the hydraulic pump which brings in a fixed or regulated flow of oil to the hydraulic cylinder, to move the piston. The piston pushes the oil in the other chamber back to the reservoir. If we assume that the oil enters from cap end, during extension stroke, and the oil pressure in the rod end / head end is approximately zero, the force *F* on the piston rod equals the pressure *P* in the cylinder times the piston area *A*:  $F=P^*A$ 

During a retraction stroke if oil is pumped into the rod end / head end and the oil from the cap end flows back to the reservoir without pressure.

The fluid pressure in the rod end is (Pull Force)/ (piston area - piston rod area):where P is the fluid pressure,  $F_p$  is the pulling force,  $A_p$  is the piston face area and  $A_r$ is the rod cross-section area.  $P = \frac{F_p}{A_p - A_r}$ 

Hydraulic cylinders are powered from pressurized hydraulic fluid, which is typically oil. The hydraulic cylinder consists of a cylinder barrel, in which a piston connected to a piston rod moves back and front. The barrel is closed on one end by the cylinder bottom (also called the cap) and the other end by the cylinder head (also called the gland) where the piston rod comes out of the cylinder. The piston has sliding rings and seals. The piston divides the inside of the cylinder into two chambers, the bottom chamber (cap end) and the piston rod side chamber (rod end / head end)

Seals on the cylinders are chosen in compliance with the ISO standards, which provides independence from seal manufacturers. seals for operating environments with higher temperatures are also offered. All manufactured cylinders are tested in compliance with ISO standard which provides for repeatable quality.

Standard cylinders have an operating pressure of 160 bars and maximum pressure is 250 bars. It enables the final damping as well. It is also possible to order ISO cylinders that are designed in compliance with your requirements (various dimensions, various speeds etc.)

Flanges, clevises, Bush, Rod eye, fork are common cylinder mounting options. The piston rod also has mounting attachments to connect the cylinder to the object or machine component that it is pushing / pulling.

The majority of our hydraulic cylinders is suitable for use within the temperature range between -25°C to 80°C. Operating pressure varies according to the product. Piston speed is very different depending upon the installation length, treatment, material and desired execution.

#### **Single Acting Hydraulic Cylinders**

Single acting cylinders are economical and the simplest design. The working mode of cylinders with single acting operation is very simple. The supply of hydraulic fluid is implemented only on one side, which is why it can be operated only unilaterally. The return movement in this procedure is usually performed by a spring and sometimes also by its own weight if the force is not too great. A cylinder with single acting operation can usually be operated using a diverting valve. Operation using correct and appropriate components is also possible.

The most important advantage of cylinders with a Single Acting operation are:

- Smaller moving force,
- Small installation length,
- Small force of the return movement.

Hydraulic cylinders with the single acting operation do not depend on electricity and are used particularly for simple tasks like operating flaps or doors, ejection devices or drawers.

#### **Plunger Type Cylinders**

A hydraulic cylinder without a piston or piston without seals is called a plunger cylinder. A plunger cylinder can only be used as a pushing cylinder. The maximum force is a piston rod area multiplied by the pressure. This means that a plunger cylinder in general has a relatively thick piston rod

#### **Double Acting Hydraulic Cylinders**

Hydraulic cylinders with double acting operation have two opposite facing piston surfaces that control the operation of the force of hydraulic liquid, i.e. usually a special hydraulic oil that enables two active moving directions. The hydraulic energy is converted through the hydraulic liquid into the mechanical energy for the movement of the pistons. The pistons usually have

### INTRODUCTION

separate connections that enable active movement in both directions. The force is thus applied in both directions and the structure of this hydraulic cylinder is very simple.

This type of cylinder with linear movement is especially suitable for use in presses and Chippers, for opening and closing drawers and for all types of raising and lowering devices. The piston rod is attached to the piston in this structure. The piston can move faster if it has a smaller surface and slower if its surface is larger. This hydraulic cylinder is used in many types of construction machinery & equipments.

#### **Tie Rod Double Acting Hydraulic Cylinders**

Tie Rod Cylinders are the most common on agricultural application, machine tools, automotive industries, transfer lines and manufacturing devices. The main feature of the tie rod cylinder is:space-saving compact design, which makes it particularly suitable for manufacturing devices., The top & bottom of the tie rod cylinder, as well as the cylinder tube are connected together via rods. Extended tie rods at the head or base of the cylinder may be used to mount the tie rod cylinder. Threaded holes and subplate mounting options are also included in the wide mounting range of the tie rod cylinder. The Tie rod cylinders use 4 long high strength threaded steel rods to hold the two end caps to the cylinder tube and are fitted that run the length of the cylinder. Small bore cylinders usually have 4 tie rods, while large bore cylinders may require many tie rods in order to retain the end caps under tremendous force produced.

The National Fluid Power Association has standardized the dimensions of hydraulic Tie rod cylinders. This enables cylinders from different manufacturers to interchange within the same mounting.

The most important advantage of Tie rod cylinder operation is:

- Compact design,
- Wide mounting range,
- Limited piston diameter,
- Limited stroke length,



#### **Connections**

The cylinders are supplied as per standard with cylindrical BSP threads and spot facing for seal rings in compliance with ISO 1179. For further information and for the order identification code, please consult our technical office.

For correct cylinder operation, fluid velocity must not exceed 0.5 m/s.

#### **Tie Rod Tightening Torque**

If the cylinder has been disassembled, re-assemble it and tighten the tie rod lock nuts cross-wise applying a gradual torque up to the value indicated in the table below. The values below refer to dry threads.

Bore (mm)	40	50	63	80	100
Tie Rod	M8x1 pitch	M12x1.5 pitch	M12x1.5 pitch	M16x1.5 pitch	M16x1.5 pitch
Torque (Nm)	20	70	70	160	160

#### Cushioning

On request, gradual & adjustable cushioning devices can be fitted in the front and / or rear ends of the cylinder without affecting overall dimensions.

The special design of the cushions ensures optimal repeatability also in the event of variations in fluid viscosity.

Cushioning devices are always recommended as they ensure impact free stopping even at high speed thus reducing pressure surges and impact transferred to the mounting supports.

For all the available bores, cushioning is adjustable by means of a needle.

Rapid piston start-up is guaranteed by the bypass valves located inside the front cushioning cone & rear cushioning ring.

### SERIES – HC/HCT/CHC

#### Tie Rod Mounting Style - (in compliance with ISO 6020-1)

L2 (ISO ME6)

Rear Flange Mounting

T4 (ISO MX5)

Front Threaded

Holes Mounting

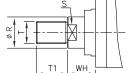


L1 (ISO ME5) Front Flange Mounting

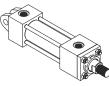


T1 (ISO MX1) Front & Back Tie Rod Mounting T2 (ISO MX2) Back Tie Rod Mounting T3 (ISO MX3) Front Tie Rod Mounting

#### Piston Rod End Type (ISO 4395) (Standard Male Thread)



	-				
Bore	Rod	т	T1	WН	s
Ø	Ø				
	18	M14x1.5	18		14
40	22	M16x1.5	22	25	17
	28	M20x1.5	28		22
	22	M16x1.5	22		17
50	28	M20x1.5	28	26	22
	36	M27x2	36		30
	28	M20x1.5	28		22
63	36	M27x2	36	33	30
	45	M33x2	45		36
	36	M27x2	36		30
80	45	M33x2	45	31	36
	56	M42x2	56		50
	45	M33x2	45		36
100	56	M42x2	56	35	50
	70	M48x2	63		60



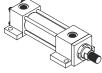
C1 (ISO MP1) Female Clevis Mounting



T5 (ISO MX6) Back Threaded Holes Mounting

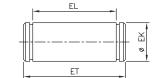


C2 (ISO MP3) Male Clevis Mounting

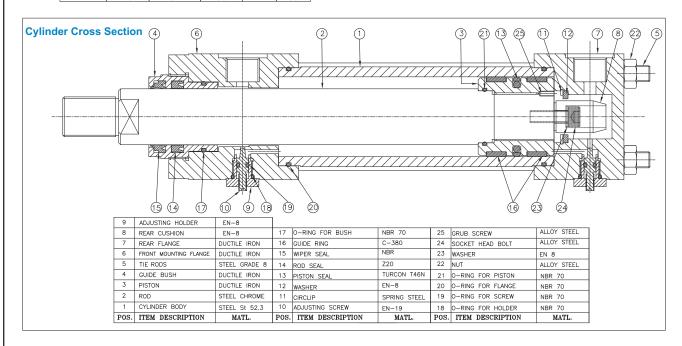


FT (ISO MS2) Side Feet Mounting

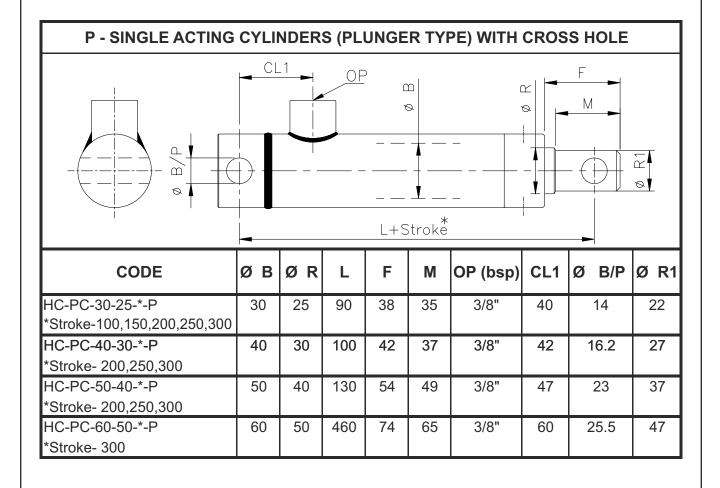
#### Female Clevis Pin (ISO 8133)



Bore ø	Rod ø	EK f8	EL +0 -0.2	ΕT
40	18 22 28	14	45	53
50 63	22 28 36	20	66	75
80	36 45 56	28	87	96
100	45 56 70	36	107	120



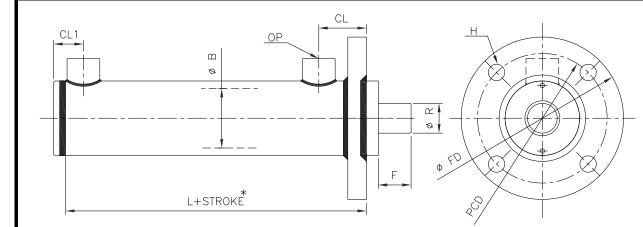
S- SINGLE ACTING STANDARD CYLINDERS (PLUNGER CYLINDER) - BASIC TYPE									
CODE	ØB	ØR	L	F	OP (bsp)	CL1			
HC-PC-40-30-*-S *Stroke- 200,250,300	40	30	130	40	3/8"	27			
				45	0/01				
HC-PC-50-40-*-S *Stroke- 200, 300	50	40	142	45	3/8"	30			



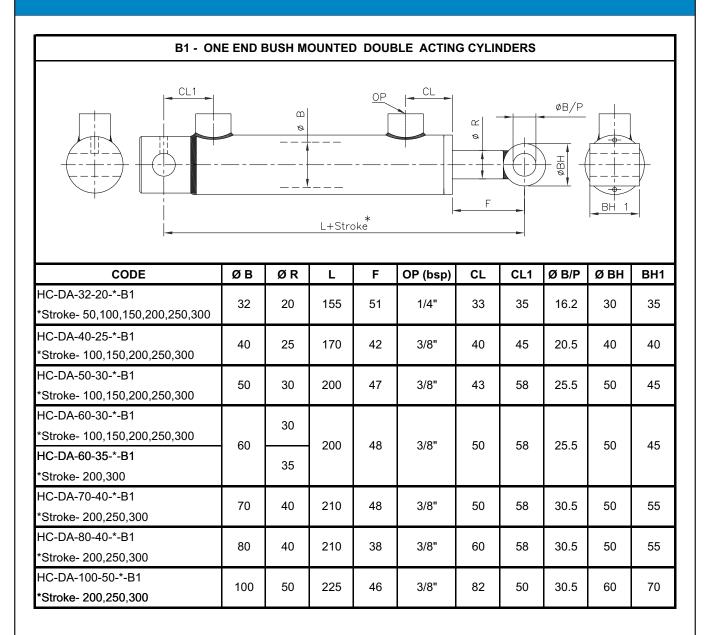
S - DOUBLE ACTING ST		ARD (	CYLIN	DER	S - BASI	C TYF	ΡE
CL1 m s L+Stro	<u>C</u> 						
CODE	ØВ	ØR	L	F	OP (bsp)	CL	CL1
HC-DA-32-20-*-S *Stroke-50,100,150,200,250,300	32	20	105	16	1/4"	33	20
HC-DA-40-20-*-S *Stroke- 100,150,200,250,300	40	20	130	22	1/4"	33	20
HC-DA-40-25-*-S *Stroke- 150,200,250,300	40	25	130	22	1/4"	40	23
HC-DA-50-25-*-S *Stroke-100,150,200,250,300	50	25	140	22	3/8"	43	26
HC-DA-50-30-*-S *Stroke-150,200,250,300	50	30	140	22	3/8"	43	26
HC-DA-60-30-*-S HC-DA-60-35-*-S *Stroke- 100,150,200,250,300 HC-DA-60-40-*-S *Stroke- 200,250,300	60	30 35 40	160	23	3/8"	50	30
HC-DA-63-40-*-S *Stroke- 200,250,300	63	40	160	23	3/8"	50	30
HC-DA-70-35-*-S *Stroke-100,150,200,250,300	70	35 40	160	23	3/8"	50	33
HC-DA-80-40-*-S *Stroke- 200,250,300	80	40 50	180	25	1/2"	60	35
HC-DA-100-50-*-S *Stroke- 200,250,300	100	50 60	210	25	1/2"	82	38

## SERIES – HC/HCT/CHC

### L1 - FLANGE MOUNTED DOUBLE ACTING STANDARD CYLINDERS



CODE	ØВ	ØR	L	F	OP (bsp)	CL	CL1	Ø FD	PCD
HC-DA-40-20-*-L1		20							
HC-DA-40-25-*-L1	40	25	101	22	1/4"	32	20	109	87
*Stroke- 100,200,300									
HC-DA-50-25-*-L1		25							
HC-DA-50-30-*-L1	50	30	112	22	3/8"	35	26	128	105
*Stroke- 100,200,300									
HC-DA-60-30-*-L1		30							
HC-DA-60-35-*-L1		35							
*Stroke- 100,200,300	60		130	23	3/8"	40	30	142	117
HC-DA-60-40-*-L1		40							
*Stroke- 200,300									
HC-DA-70-35-*-L1		35	130						
*Stroke- 100,200,300	70	- 55	130	23	3/8"	40	33	162	127
HC-DA-70-40-*-L1	10	40	260	23	3/0	40	33	102	121
*Stroke- 200,300		40	200						
HC-DA-80-40-*-L1			150						
HC-DA-80-50-*-L1	80	40	250	25	1/2"	50	35	181	149
*Stroke- 100,200,300									
HC-DA-100-50-*-L1		50							
*Stroke- 100,200,300	100	50	178	25	1/2"	70	38	194	162
HC-DA-100-60-*-L1	100	60	170	20	1/2	70	30	194	102
*Stroke- 100,200,300		00							



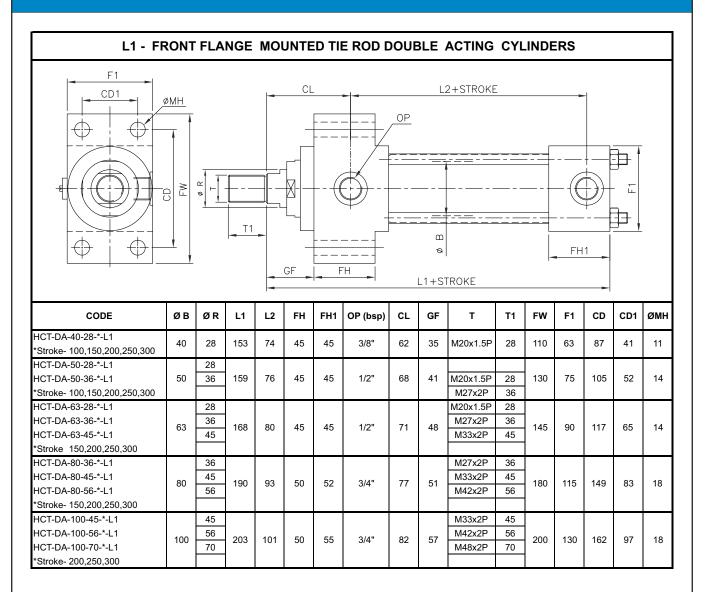
B2 - E	зотн	END B	USH MC	DUNTED	DOUI	BLE ACT	ING C	YLIND	ERS			
			۳ م 	 Z+Stroi L+Strok					ØB/F			
CODE	ØВ	ØR	L	Z	F	OP(bsp)	CL	CL1	ØB/P	Ø BH	BH1	BH2
HC-DA-40-20-*-B2 *Stroke- 100,150,200,250,300	40	20	165	130	39.5	1/4"	33	40.5	16.25	35	30	60
HC-DA-40-25-*-B2 *Stroke- 150,200,250,300	40	25	165	130	39.5	1/4"	40	40.5	16.25	35	30	60
HC-DA-50-25-*-B2 *Stroke- 100,150,200,250,300	50	25	180	140	43	3/8"	43	46	20.25	40	40	70
HC-DA-50-30-*-B2 *Stroke- 150,200,250,300	50	30	180	140	43	3/8"	43	46	20.25	40	40	70
HC-DA-60-30-*-B2 *Stroke- 100,150,200,250,300		30										
HC-DA-60-35-*-B2 *Stroke- 100,150,200,250,300	60	35	210	160	48	3/8"	50	55	25.25	50	50	80
HC-DA-60-40-*-B2 *Stroke- 200,250,300		40										
HC-DA-63-40-*-B2 *Stroke- 200,250,300	63	40	210	160	48	3/8"	50	55	25.25	50	50	80
HC-DA-70-35-*-B2 *Stroke- 100,150,200,250,300	70	35	210	160	48	3/8"	50	58	25.25	50	50	90
HC-DA-80-40-*-B2 HC-DA-80-50-*-B2 *Stroke- 200,250,300	80	40 50	240	180	55	1/2"	60	65	30.25	60	60	110
HC-DA-100-50-*-B2 *Stroke- 200,250,300 HC-DA-100-60-*-B2	- 100	50	280	210	60	1/2"	82	73	40.25	70	70	130
HC-DA-100-60-*-B2 *Stroke- 300		60										

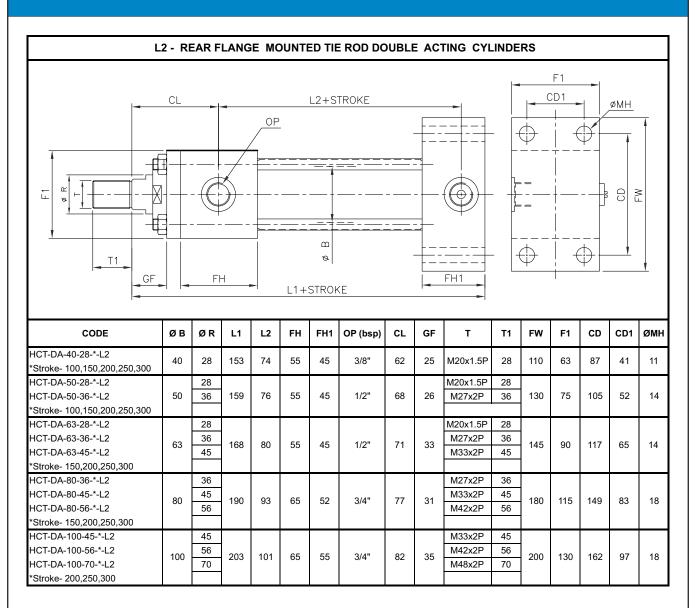
# SERIES – HC/HCT/CHC

## F - BOTH END FORK MOUNTED DOUBLE ACTING CYLINDERS

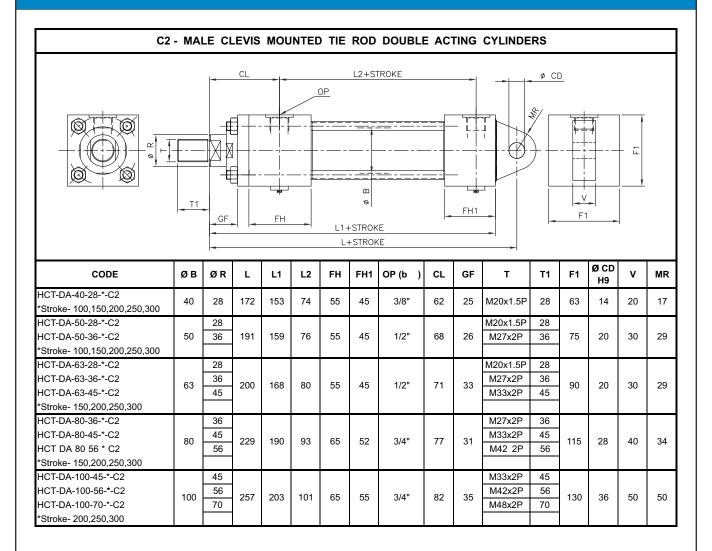
		  Z+Stro L+Stro					ØB/F				
CODE	ØВ	ØR	L	z	OP (bsp)	М	CL	CL1	Ø B/P	SQ- BH	w
HC-DA-32-20-*-F *Stroke- 50,100,150,200,250,300	32	20	185	105	1/4"	40	33	60	20.25	40	20
HC-DA-40-20-*-F *Stroke- 100,150,200,250,300	40	20	210	130	1/4"	40	33	60	20.25	40	20
HC-DA-50-25-*-F *Stroke- 100,150,200,250,300 HC-DA-50-30-*-F *Stroke- 150,200,250,300	- 50	25 30	230	140	3/8"	45	43	71	25.25	50	25
HC-DA-60-30-*-F *Stroke- 100,150,200,250,300 HC-DA-60-35-*-F *Stroke- 100,150,200,250,300 HC-DA-60-40-*-F	60	30 35 40	260	160	3/8"	50	50	80	30.25	60	30
*Stroke- 200,250,300 HC-DA-63-40-*-F *Stroke- 200,250,300	63	40	260	160	3/8"	50	50	80	30.25	60	30
HC-DA-70-35-*-F *Stroke- 100,150,200,250,300 HC-DA-70-40-*-F *Stroke- 200,250,300	- 70	35 40	260	160	3/8"	50	50	83	30.25	60	30
HC-DA-80-40-*-F *Stroke- 200,250,300 HC-DA-80-50-*-F *Stroke- 200,250,300	- 80	40 50	290	180	1/2"	55	60	90	35.25	70	35

E - BOTH END F	ROD- E	YE N	IOUN	red i	DOUB	LE ACT	ING	CYLIN	DERS	5	
CL1											
CODE	ØB	ØR	L	z	F	OP (bsp)	CL	CL1	Ø B/P	BH	BH1
HC-DA-32-20-*-E *Stroke- 50,100,150,200,250,300	32	20	165	105	46	1/4"	33	50	20.25	45	25
HC-DA-40-20-*-E *Stroke- 100,150,200,250,300	40	20	190	130	52	1/4"	33	50	25.25	50	25
HC-DA-40-25-*-E *Stroke- 150,200,250,300	40	25	190	130	52	1/4"	40	50	25.25	45	25
HC-DA-50-25-*-E *Stroke- 100,150,200,250,300 HC-DA-50-30-*-E *Stroke- 150,200,250,300	- 50	25 30	210	140	57	3/8"	43	61	25.25	50	30
HC-DA-60-30-*-E *Stroke- 100,150,200,250,300 HC-DA-60-35-*-E *Stroke- 100,150,200,250,300 HC-DA-60-40-*-E	60	30 35 40	250	160	68	3/8"	50	76	30.25	60	35
*Stroke- 200,250,300 HC-DA-63-40-*-E *Stroke- 200,250,300	63	40	250	160	68	3/8"	50	76	30.25	60	35
HC-DA-70-35-*-E *Stroke- 100,150,200,250,300 HC-DA-70-40-*-E *Stroke- 200,250,300	70	35 40	250	160	68	3/8"	50	78	30.25	60	35
HC-DA-80-40-*-E *Stroke- 200,250,300 HC-DA-80-50-*-E *Stroke- 200,250,300	- 80	40 50	290	180	80	1/4"	60	90	30.5	70	40





C	:1 - FI	EMAL	E CLE	EVIS	MOUN	ITED	TIE F	ROD DOI	JBLE	ACT	NG CYLI	NDER	s				
CL L2+STROKE CL UP CL UP CL UP CL UP CL CL CL CL CL CL CL CL CL CL																	
CODE	ØВ	ØR	L	L1	L2	FH	FH1	OP (bsp)	CL	GF	т	T1	F1	Ø CD H9	v	V1	MR
HCT-DA-40-28-*-C1 *Stroke- 100,150,200,250,300	40	28	172	153	74	55	45	3/8"	62	25	M20x1.5P	28	63	14	20	42	17
HCT-DA-50-28-*-C1 HCT-DA-50-36-*-C1 *Stroke- 100,150,200,250,300	50	28 36	191	159	76	55	45	1/2"	68	26	M20x1.5P M27x2P	28 36	75	20	30	62	29
HCT-DA-63-28-*-C1 HCT-DA-63-36-*-C1 HCT-DA-63-45-*-C1 *Stroke- 150,200,250,300	63	28 36 45	200	168	80	55	45	1/2"	71	33	M20x1.5P M27x2P M33x2P	28 36 45	90	20	30	62	29
HCT-DA-80-36-*-C1 HCT-DA-80-45-*-C1 HCT-DA-80-56-*-C1 *Stroke- 150,200,250,300	80	36 45 56	229	190	93	65	52	3/4"	77	31	M27x2P M33x2P M42x2P	36 45 56	115	28	40	83	34
HCT-DA-100-45-*-C1 HCT-DA-100-56-*-C1 HCT-DA-100-70-*-C1 *Stroke- 200,250,300	100	45 56 70	257	203	101	65	55	3/4"	82	35	M33x2P M42x2P M48x2P	45 56 70	130	36	50	103	50



# **PUSH & PULL FORCE**

		PUSH & PULL FORCE IN VARIOUS BAR PRESSURE										
CYLINDER BORE / ROD	50 bar	75 bar	100 bar	125 bar	160 bar	50 bar	75 bar	100 bar	125 bar	160 bar		
BOILTROD		PUSH	I FORCE	IN kN		PULL FORCE IN kN						
30/25	3.5	5.3	7.0	8.8	11.3	1.0	1.6	2.1	2.6	3.4		
32/20	4.0	6.0	8.0	10.0	12.8	2.4	3.6	4.9	6.1	7.8		
40/20	6.3	9.5	12.6	15.8	20.1	4.7	7.0	9.4	11.7	15		
40/25	6.3	9.5	12.6	15.8	20.1	3.8	5.7	7.7	9.6	12.2		
40/30	6.3	9.5	12.6	15.8	20.1	2.7	8.4	5.4	6.8	8.7		
50/25	9.8	14.7	19.6	24.5	31.4	7.3	11	14.7	18.3	23.5		
50/30	9.8	14.7	19.6	24.5	31.4	6.3	9.4	12.6	15.7	20.1		
50/40	9.8	14.7	19.6	24.5	31.4	3.5	5.3	7.0	8.8	11.3		
60/30	14.2	21.2	28.3	35.4	45.2	10.6	15.9	21.2	26.5	33.9		
60/35	14.2	21.2	28.3	35.4	45.2	9.3	14	18.7	23.3	29.8		
60/40	14.2	21.2	28.3	35.4	45.2	7.8	11.7	15.7	19.6	25.1		
60/50	14.2	21.2	28.3	35.4	45.2	4.3	6.4	8.6	10.7	13.8		
63/40	15.5	23.3	31.1	38.9	49.8	9.3	13.9	18.6	23.2	29.7		
70/35	19.2	28.8	38.5	48.1	61.5	14.4	21.6	28.9	36.1	46.1		
70/40	19.2	28.8	38.5	18.1	61.5	12.9	19.4	25.9	32.3	41.4		
80/40	25.1	37.7	50.3	62.8	80.4	18.8	28.2	37.7	47.1	60.3		
80/50	25.1	37.7	50.3	62.8	80.4	15.3	22.9	306	38.2	49		
100/50	39.2	58.8	78.5	98.1	125.6	29.4	44.1	58.9	73.6	94.2		
100/60	39.2	58.8	78.5	98.1	125.6	25.1	37.6	50.2	62.8	80.4		

### **MAINTENANCE & TROUBLE SHOOTING**

- a. <u>External Leakage.</u> If a cylinder's end caps are leaking, tighten them. If the leaks still do not stop, replace the gasket. If a cylinder leaks around a piston rod, replace the packing. Make sure that a seal lip faces toward the pressure oil. If a seal continues to leak, check points e through I.
- b. Internal Leakage. Leakage past the piston seals inside a cylinder can cause sluggish movement or settling under load. Piston leakage can be caused by worn piston seals or rings or scored cylinder walls. The latter may be caused by dirt and grit in the oil.
- c. <u>Creeping Cylinder</u>. If a cylinder creeps when stopped in midstroke, check for internal leakage (point b). Another cause could be a worn control valve.
- d. <u>Sluggish Operation</u>. Air in a cylinder is the most common cause of sluggish action. Internal leakage in a cylinder is another cause. If an action is sluggish when starting up a system, but speeds up when a system is warm, check for oil of too high a viscosity. If a cylinder is still sluggish after these checks, test the whole circuit for worn components.
- e. Loose Mounting. Pivot points and mounts may be loose. The bolts or pins may need to be tightened, or they may be worn out. Too much slop or float in a cylinder's mountings damages the piston-rod seals. Periodically check all the cylinders for loose mountings.
- f. Misalignment. Piston rods must work in-line at all times. If they are side-loaded, the piston rods will be galled and the packings will be damaged causing leaks. Eventually, the piston rods may be bent or the welds broken.
- g. Lack of Lubrication. If a piston rod has no lubrication, a rod packing could seize, which would result in an erratic stroke, especially on single-acting cylinders.
- h. Abrasives on a Piston Rod. When a piston rod extends, it can pick up dirt and other material. When it retracts, it carries the grit into a cylinder, damaging a rod seal. For this reason, rod wipers are often used at the rod end of a cylinder to clean the rod as it retracts. Rubber boots are also used over the end of a cylinder in some cases. Piston rods rusting is another problem. When storing cylinders, always retract the piston rods to protect them. If you cannot retract them, coat them with grease.
- I. Burrs on a Piston Rod. Exposed piston rods can be damaged by impact with hard objects. If a smooth surface of a rod is marred, a rod seal may be damaged. Clean the burrs on a rod immediately, using crocus cloth. Some rods are chrome-plated to resist wear. Replace the seals after restoring a rod surface.
- j. Air Vents, Single-acting cylinders (except ram types) must have an air vent in the dry side of a cylinder. To prevent dirt from getting in, use different filter devices. Most are self-cleaning, but inspect them periodically to ensure that they operate properly.

NOTE: When repairing a cylinder, replace all the seals and packings before reassembly.

#### **Trouble Shooting Guidelines**

Cylinder failure can occur for many reasons. This cylinder Trouble Shooting Guideline can be used to analyze the potential reasons for cylinder failure and establish corrective actions

Symptom	Possible Causes	Remedies
Piston rod scored	Contamination of the oil Contamination of the gland bearing	Flush entire hydraulic system Change all filters
Cylinder bore scored	Contamination of the oil Piston bearing failure Damage cylinder barrel	Flush entire hydraulic system Change all filters Check piston head bearing Replace cylinder barrel
Bent piston rod	Operation problem: possible overload Outside impact Under specification of piston rod	Check operation parameters Increase rod specification
Split weld on base and ports	Shock loading Poor original weld	Check operation parameters Machine off & re-weld correctly
Rod worn on one side	Lack of bearing support Too much side load Rod too small	Increase bearing area Change operation Increase rod size Incorporate external guides
Gland blown out	Possible intensification of internal pressure Threads worn Deformation of cylinder tube	Check hydraulic valve operation Check threads Check tube for ovality & thread wear
Leaking from around the gland O.D.	o-ring failure cracked gland	Check clearances Fit back-up ring to o-ring Crack test gland
Piston rod pitting	Corrosion Piston rod damage	Upgrade to anti-corrosive material Specification. Protect rod from weather. Check rod for nicks or scratches that could cause seal damage or allow oil leakage.
Barrel internally corroded	Water in the oil	Change oil Protect from water ingress

### **TROUBLE SHOOTING**

Symptom	Possible Causes	Remedies	
Piston rod will not retract	Internal leakage Port blockage	Strip and inspect piston head & tube Check ports and pipes for blockages Check valve operation	
Regular seal leakage	Incorrect seals fitted Seal grooves corroded or marked Air trapped in the oil Incorrect metalwork clearances Contamination of the oil Seals fitted incorrectly Seal housing sizes incorrect	Check seal compatibility with conditions Check all seal grooves for marking & corrosion Make sure the cylinder is bled correctly Check oil for contamination Check condition of all running surfaces	
	Rod seal leaking	Replace the seal, If contamination caused seal to wear may be caused by external as well internal contaminants.	
Cylinder is getting hot	Internal leakage	Test for internal pressure bypass Strip cylinder and inspect piston head & bore of the tube Remove the piston and check the internal seal	
Rod kick up at end of stroke	Internal bearing wear Incorrect alignment of piston to gland	Strip and inspect piston head & gland bearings Check alignment of piston to the gland	
Loss of power	Internal leakage Hydraulic pump failure Valve settings incorrect	Strip and inspect piston head & tube Check pump & valves	
Slip stick or juddering	Lack of lubricant for the gland bearing & seals	Rough surface finishes on rod or tube Lack of hydraulic pressure Vacuum or air entrapment Bearing tolerances too tight Seals too tight	
Erratic Action	Pilot control pressure low Air in system	Control line may be too small or melting choke valve not working properly. Bleed air and check for leaks. Check to see that oil intake is well below the surface of the oil in the reservoir. Check pump packing and line connections on the intake side by pouring hydraulic oil over suspected leak. If the noise stops, the leak has been located. Tighten joints or change packing or gaskets packing or gaskets where necessary.	

#### **General Notes**

<u>Maximum pressure</u> – Microtact Hydraulic Cylinders are designed in accordance with standards for a dynamic continuous pressure of 160 bar for all mounting types. Under certain conditions, a higher pressure may be permitted. To confirm this, we require a detailed application, description on the basis of a technical data of a regenerative circuit or a meter-out throttle, pressure intensification must be taken into account.

**Minimum pressure –** Depending on the application, a certain minimum pressure is required to ensure correct operation of the cylinder. Under no -load condition, a minimum pressure of 10 bar is recommended for single rod cylinders. In the case of lower pressure, please consult us.

Installation of cylinder – The cylinder may only be installed or the piston rod end screwed into the machine part or into a selfaligning clevis while the cylinder is depressurized.

#### Hydraulic Cylinder Formula

Calculate	Formula	Symbolic	
Cylinder area	Area= $\pi$ Radius <sup>2</sup> (Inches)	Area= πr <sup>2</sup>	
(In Sq. Inch)	Area= (P/4) x Diameter <sup>2</sup> (Inches)	A = $(\pi D^2)/4$ or A= 0.785D <sup>2</sup>	
Cylinder Force (In Pounds, Push or Pull)	Area=Pressure (psi) x Net Area (sq. inch)	F = psi x A or F = PA	
Cylinder Velocity or Speed (In Feet/Second)	Velocity = 231xFlow Rate (GPM) ÷ 12 x 60 x Net Area (sq. in.)	V = 231 Q÷720A or V = 0.3208 Q ÷ A	
Cylinder Volume Capacity	Volume = $\pi$ x Radius <sup>2</sup> (in) x Stroke (in) ÷ 231	$V = (\pi r^2 L) \div 231$	
(In Gallons of Fluid)	Volume = Net Area (sq. in) x stroke (in.) ÷ 231	V = (AL) ÷ 231	
Cylinder Flow Rate (In Gallons / Minute)	Flow Rate = 12 x 60 x Velocity (Ft / Sec.) x Net Area (sq. in.) / 231	Q = (270vA) 231 or Q = 3.117vA	

# Hydraulic Formulas

Horsepower =GPM x psi	overall Efficiency : overall efficiency = <u>OUTPUT HP</u> x 100 INPUT HP x 100			
Torque:	olumetric Efficiency:			
Torque (lb. in.) = <u>HP x 63025</u> RPM	olumetric efficiency (pump) = <u>OUTPUT_GPM</u> × 100 THEORETICAL_GPM			
V	olumetric efficiency (motor) = <u>THEORETICAL GPM</u> x 100 INPUT GPM			
Flow: Flow(gpm)= <u>CUIN./REV.x RPM</u> 231				
<b>CONVERSION FACTORS:</b> 1 hp = 33,000 ft. lbs. per minute 1 hp = 42.4 btu per minute 1 hp = 0.746 kwhr (kilowatt hours) 1 U. S. gallon = 231 cubic inches.				
Pipe volume varies as the square of the diame gallons = 0.0034 D <sup>2</sup> L	ter; volume in			
where: D = inside diameter of pipe in inches L = length in inches. Velocity in feet per second = $0.408 \text{ x}$	flow (gpm)			
where: D = inside diameter of pipe in inches.	D2			
Atmospheric pressure at sea level = 14.7 psi Atmospheric pressure decreases approximately thousand feet of elevation up to 23,000 feet.	0.41 psi for each one			
Pressure (psi) = feet head x 0.433 x specific gravity. Specific gravity of oil is approximately 0.85. Thermal expansion of oil is approximately 1 cu. in. per 1 gal. per $10^{\circ}$ F rise in temperature.				
<b>Practical hydraulic formulae</b> Geometric flow rate (I/min) (pumps and motors)	= <u>Geometric displacement (cm<sup>3</sup>/r) x shaft speed (r/min)</u> 1000			
Theoretical shaft torque (Nm) (pumps and motors)	= <u>Geometric displacement (cm<sup>3</sup>/r) x pressure (bar)</u> 20 $\pi$			
Shaft power (kW)	= <u>Torque at shaft (Nm)_x_shaft speed (r/min)</u> 9550			
Hydraulic power (kW)	= <u>Flow rate (I/min) x pressure (bar)</u> 600			
Heat equivalent of hydraulic power (kJ/mir	min) = <u>Flow rate (I/min) x pressure (bar)</u>			
Geometric flow rate (I/min) (cylinders)	= <u>Effective area (cm<sup>2</sup>) x piston speed (m/min)</u>			
Theoretical force (N) (cylinders)	=Effective area (cm <sup>2</sup> ) x pressure (bar) x 10			
Velocity of fluid in pipe (m/s)	= <u>Flow rate (I/min) x 21,22</u> D <sup>2</sup>			
	where D = inside diameter of pipe in millimeters.			

# **Conversion Factors**

To convert —		Into     Multiply by			
Into ৰ		To convert -	<b></b>	Divide by	
Unit	Symbol	Unit	Symbol	Factor	
Atmospheres	Atm	bar	bar	1.013250	
BTU / hour	Btu/h	kilowatts	kW	0.293071 x 10 <sup>-3</sup>	
ubic centimeters cm <sup>3</sup>		liters	L	L 0.001	
Cubic centimeters	cm <sup>3</sup>	milliliters	ml	1.0	
Cubic feet	ft <sup>3</sup>	cubic meters	m <sup>3</sup>	0.0283168	
Cubic feet	ft <sup>3</sup>	litres	L	28.3161	
Cubic inches	in³	cubic centimeters	cm <sup>3</sup>	16.3871	
Cubic inches	in <sup>3</sup>	liters	L	0.0163866	
Degrees (angle)	0	radians	rad	0.0174533	
Fahrenheit	°F	Celsius	٥°C	°C = 5(°F - 32)/9	
Feet	ft	meters	m	0.3048	
Feet of water	ftH₂0	bar	bar	0.0298907	
Fluid ounces. UK	Uk fl oz	cubic centimeters	cm <sup>3</sup>	28.413	
Fluid ounces. US	Us fl oz	cubic centimeters	cm <sup>3</sup>	29.5735	
Foot pounds f.	ft lbf	joules	J	1.35582	
Foot pounds / minute	ft lbf/min	watts	W	81.3492	
Gallons. UK	UK gal	liters	L	4.54596	
Gallons. US	US gal	liters	L	3.78531	
Horsepower	hp	kilowatts	- kW	0.7457	
Inches of mercury	in Hg	millibar	mbar	33.8639	
Inches of water	in H <sub>2</sub> O	millibar	mbar	2.49089	
Inches	in 1120	centimeters	cm	2.54	
Inches	in	millimeters	mm	25.4	
Kilogramm force	kgf	newtons	N	9.80665	
Kilogramm f. meter	kgf m	newton meters	Nm	9.80665	
Kilogramm f./sq centimeter	kgf/cm <sup>2</sup>	bar	bar	0.980665	
Kilopascals	kgi/cm kPa	bar	bar	0.01	
Kilopound	-	newtons	N	9.80665	
Kilopound meters	kp kp m	newton meters	Nm	9.80665	
Kilopound/square centimeter	kp m kp/cm <sup>2</sup>	bar	bar	0.980665	
Microinches	kp/cm in	microns	μm	0.0254	
Millimetres of mercury		millibar	mbar	1.33322	
Millimeters of water	mm hg	millibar		0.09806	
Newtons/square centimeter	mm H₂O N/cm²	bar	mbar	0.09800	
Newtons/square meter		bar	bar bar	10 <sup>-5</sup>	
•	N/m <sup>2</sup>			10 <sup>-5</sup>	
Pascals (newtons/sq meter)	Pa	bar	bar		
Pounds (mass) Pounds / cubic foot	lb	kilograms kilograms/ cubic meter	kg	0.4536 16.0185	
Pounds / cubic foot	Ib/ft <sup>3</sup>		kg/m <sup>3</sup>		
	lb/in <sup>3</sup>	kilograms/ cubic centimeter	kg/cm <sup>3</sup>	0.0276799	
Pounds force	lbf	newtons	N	4.4822	
Pounds f. feet	lbf ft	newton meters	Nm	1.35582	
Pounds f. inches	lbf in	newton meters	Nm	0.112985	
Pounds f. / square inch	lbf/in <sup>2</sup>	bar .	bar	0.06894	
Revolutions/minute	r/min	radians/second	rad/s	0.104720	
Square feet	ft <sup>2</sup>	square meters	m <sup>2</sup>	0.092903	
Square inches	in²	square meters	m²	6.4516 x 10 <sup>-4</sup>	
Square inches Square inches	in² in²	square meters square centimeters	m <sup>2</sup> cm <sup>2</sup>	6.4516 x 1 6.4516	

Fluid power equipvalents 1 bar =  $10^{5}$  N/m<sup>2</sup> 1 bar = 10 N/cm<sup>2</sup> = 1 dN/mm<sup>2</sup> 1 pascal = 1 N/m 1 litre = 1000 Cm<sup>3</sup> 1 centistoke (cSt) = 1 mm<sup>2</sup>/S 1 joule = 1 wattsecond (Ws) Hertz (Hz) = cycles/second

# Prefixes denoting decimal multiples or sub-multiples

For multiples

x10 <sup>12</sup>	tera	Т
x10 <sup>9</sup>	giga	G
x10 <sup>6</sup>	mega	Μ
x10 <sup>3</sup>	kilo	k
x10 <sup>2</sup>	hecto	h
x10	deca	da

For submultiples

x10 <sup>-1</sup>	deci	d
x10 <sup>-2</sup>	centi	С
x10 <sup>-3</sup>	milli	m
x10 <sup>-6</sup>	micro	μ
x10 <sup>-9</sup>	nano	n
x10 <sup>-12</sup>	pico	р
x10 <sup>-15</sup>	femto	f
x10 <sup>-18</sup>	atto	а



# **ISO/ANSI BASIC SYMBOLS**

For Fluid Power Equipments And Systems

Lines		Pumps		<b>Miscellaneous Uni</b>	Miscellaneous Units	
Line, Working (Main) Line, Pilot		Hydraulic Pump Fixed Displacement		Electric Motor	M	
(For Control) Line, Liquid Drain		Variable		Accumulator, Spring		
Hydraulic Flow, Direction of Pneumatic		Displacement		Loaded	$\bigcirc$	
Lines Crossing	0R			Accumulator, Gas Charged		
Lines Joining Lines With Fixed Restriction		Hydraulic Motor Fixed Displacement		Heater		
Line, Flexible Station, Testing, Measurement or Power Take-Off	<ul><li>✓</li><li>✓</li></ul>	Variable Displacement	$\langle \rangle$	Cooler		
Variable Component (run arrow through symbol at 45°		Cylinder, Single Acting		Temperature Controller		
Pressure Compen-		Cylinder, Double Acting		Filter, Strainer		
sated Units (arrow parallel to short side of symbol)		Single End Rod		Pressure Switch	<b>/</b> //////////////////////////////////	
Temperature cause or Effect		Double End Rod		Pressure Indicator		
Vented Reservoir Pressurized		Adjustable Cushion advance Only		Temperature Indicator		
Line, To Reservoir Above Fluid Level	L			Component Enclosure		
Below Fluid Level Vented Manifold		Differential Piston		Direction of Shaft Rotation (assume arrow on near side		
	Ľ			of shaft)		



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